Beam Window Analysis Muon Collider Project Experiment 951

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E951 Window Analysis Overview

Proton Beam Structure

- 15 TP, 24 GeV, Gaussian profile
- smallest beam spot size assumed = 1mm RMS sigma
- pulse structure and length

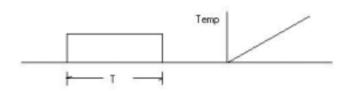
• Beam/window interactions

- A3 Line Windows
- E951 Target enclosure windows
- thermal shock
- material behavior
- Benchmark analysis and simulation verification using experiment 951 targets

Proton Beam Structure

Proton Beam Structure

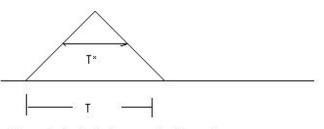
- 15 TP, 24 GeV, Gaussian profile
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T = pulse length

Proton distribution uniform in time Beam profile is Gaussian

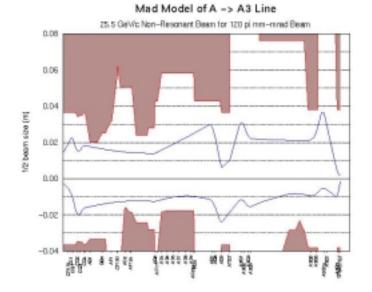
Energy deposition (thermal input rate) constant

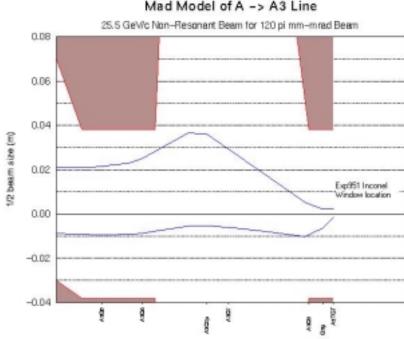


Proton distribution in time space (real beam !!) Beam = Gaussian in space

T for AGS beam ~ 100 ns T* 30 ~ 40 ns

A3 Line Beam Size

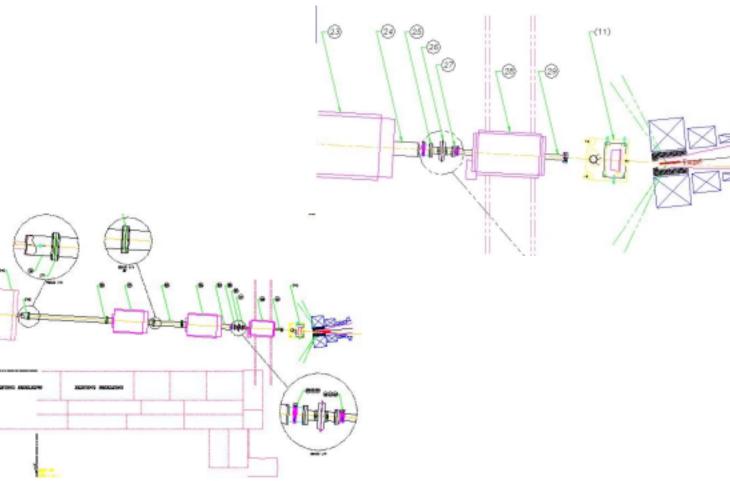




Mad Model of A -> A3 Line

A3 Line

• A3 Line near E951



A3 Line Windows

- Material used is different series Aluminum
 - 5052 series available in 3-mil thickness
- Concern is the TP per pulse coupled with a small spot size
- Experience from previous experiments showed good window response
 - an order of magnitude higher in single-pulse TP for E951

E951 Enclosure Windows

• Family of materials assessed

- INVAR
- SS316
- Inconel-718

• Beam spot size same as on target

- Estimates based on 1mm RMS sigma
- Beam spot may actually be SMALLER !!
- E951 window positioning
- Window material monitoring and failure verification

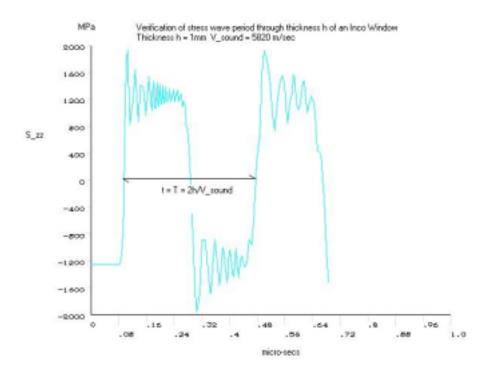
Background on Thermal Shock and Window Failure Estimation

- Quasi-static thermal stress from energy deposition is a 3-D affair no matter how thin the window is
 - directional stress in 3-D world = E*alpha*DT/1-2*pr
- Two-dimensional simplification of a thin structure does not quite apply
 - directional stress in 2-D = E*alpha*DT/1-pr

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1-D = E*alpha*DT
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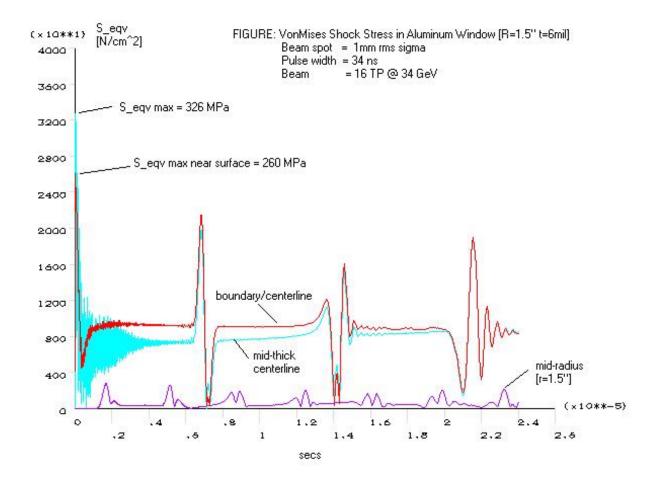
- Of concern is NOT the level of directional stress but the deviation from the hydrostatic state of stress (VonMises stress)
- Directional stresses are coupled through the Poisson's ratio
 - dynamic changes in one direction affect all others
- Build-up of thermal stress in the course of proton pulse
- Propagation and attenuation of shock or dynamic stress

Background on Thermal Shock and Window Failure Estimation

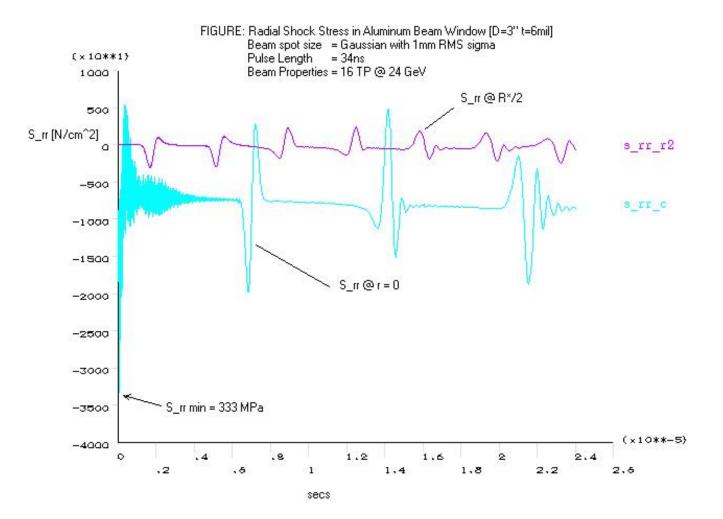


- Based on 3-D stress state the effect of throughthickness RINGING is accounted
- Its effect is **dominant** in the response of the heated window region
- Governed principally by the propagation of stress waves in 1-D space
 - stress(t) = f(x-ct) + f(x-ct) [c = speed of sound in material]
 - period of ringing = 2*h/c [h = window thickness]

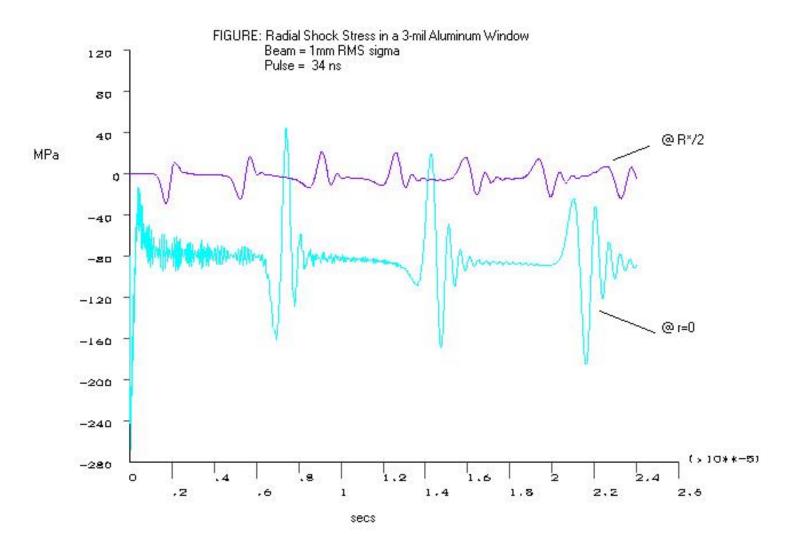
Thermal Shock Analysis of the Exp951 Windows A3 Line 3-inch diameter and 6-mil thick Aluminum Window



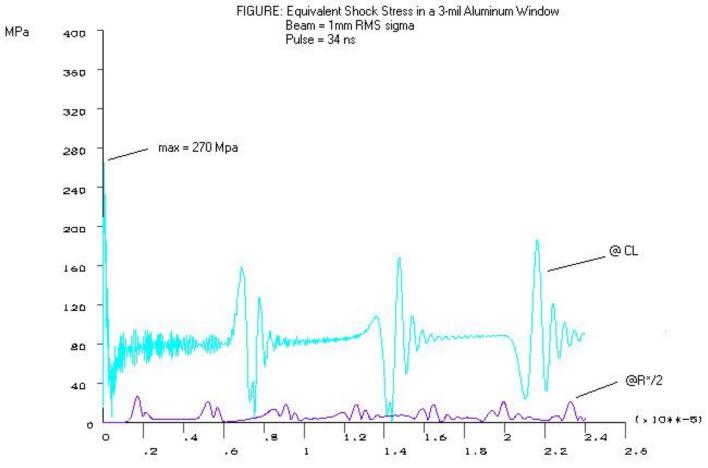
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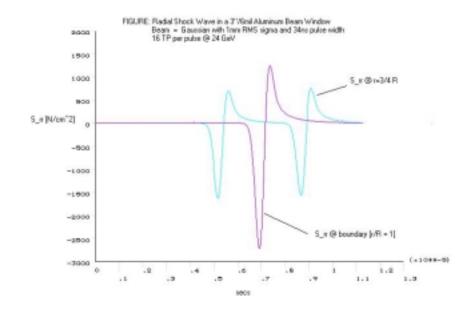
Thermal Shock Analysis of the Exp951 Windows A3 Line 3" 3-mil Aluminum Window

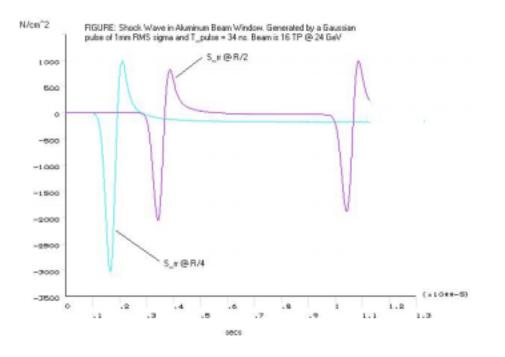


Thermal Shock Analysis of the Exp951 Windows A3 Line 3-inch diameter and 3-mil thick Aluminum Window

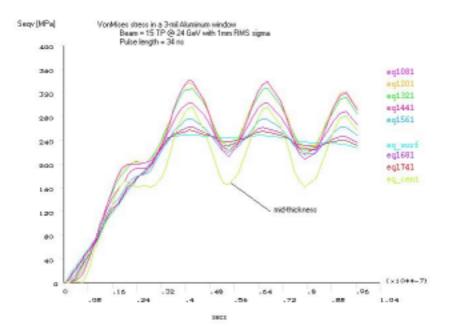


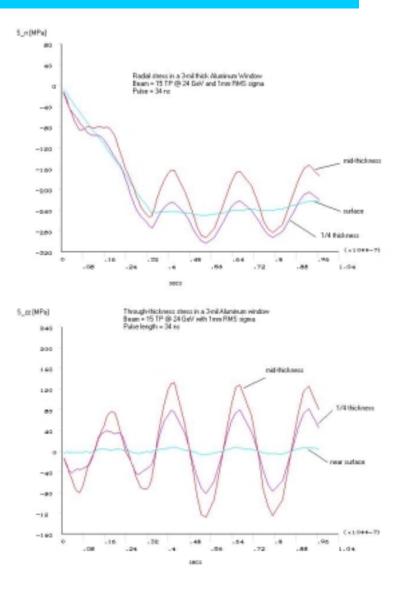
Thermal Shock Analysis of the Exp951 Windows A3 Line 3-inch diameter ; 6-mil Aluminum Window



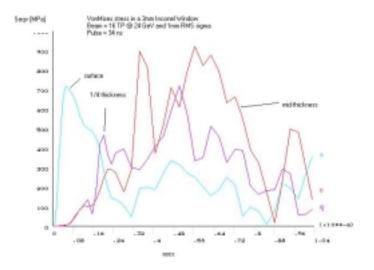


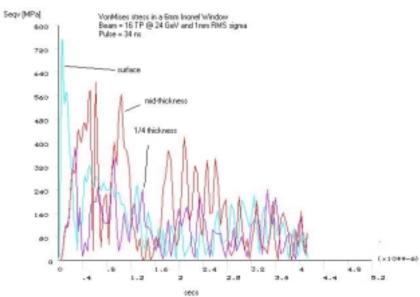
Thermal Shock Analysis of the Exp951 Windows A3 Line 3-inch diameter ; 3-mil Aluminum Window

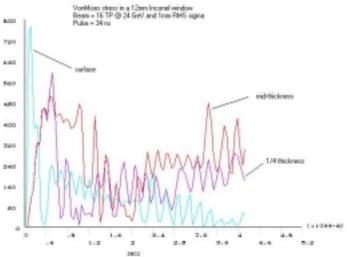




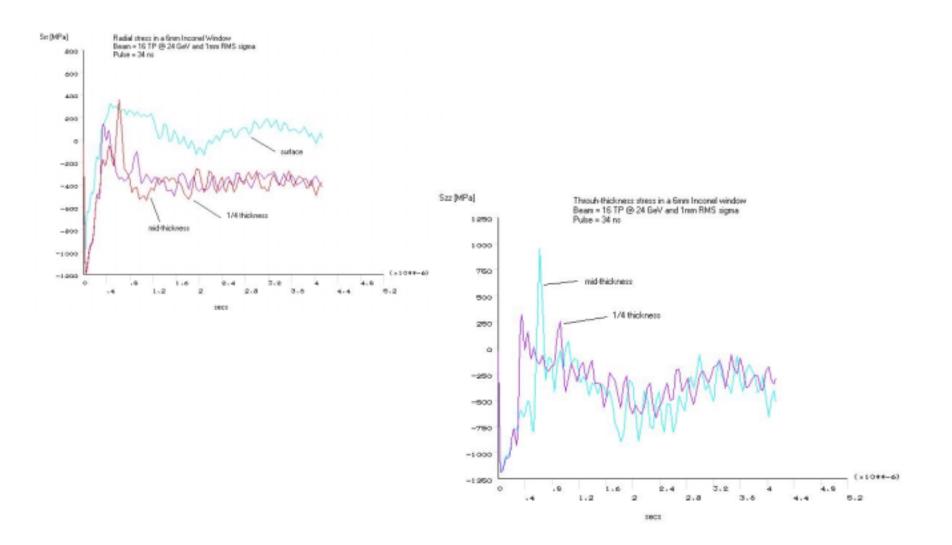
Thermal Shock Analysis of the Exp951 Windows Inconel-718 Window Analysis





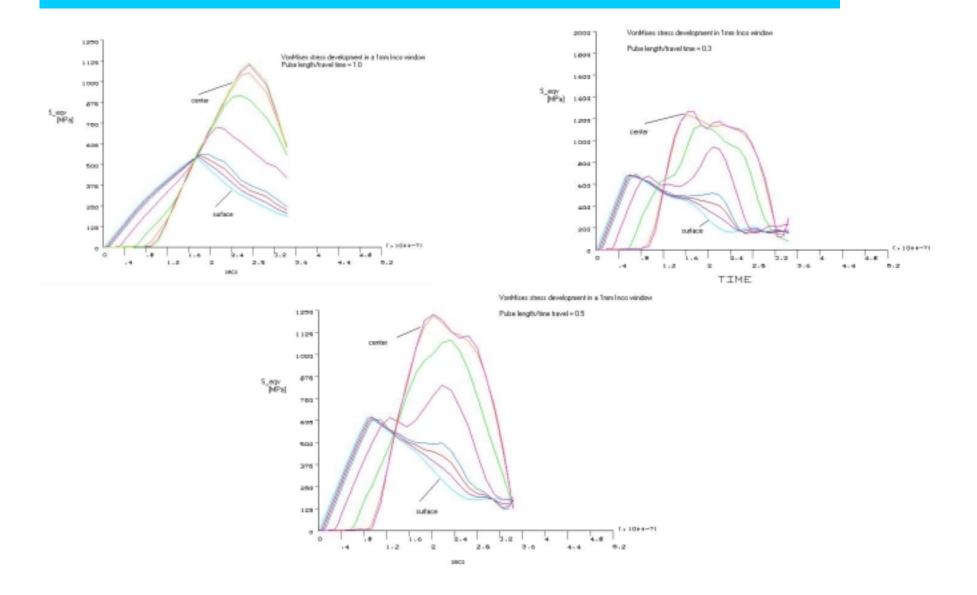


Thermal Shock Analysis of the Exp951 Windows Inconel-718 Window Analysis

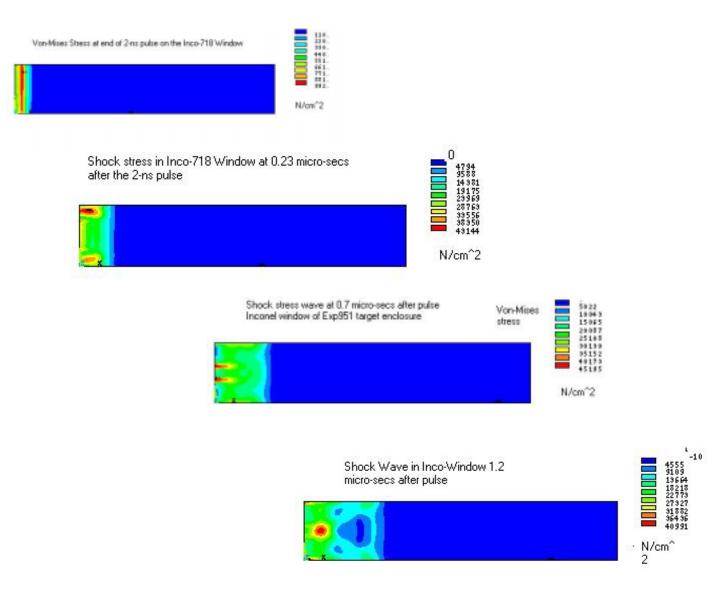


Inconel-718 Window Analysis

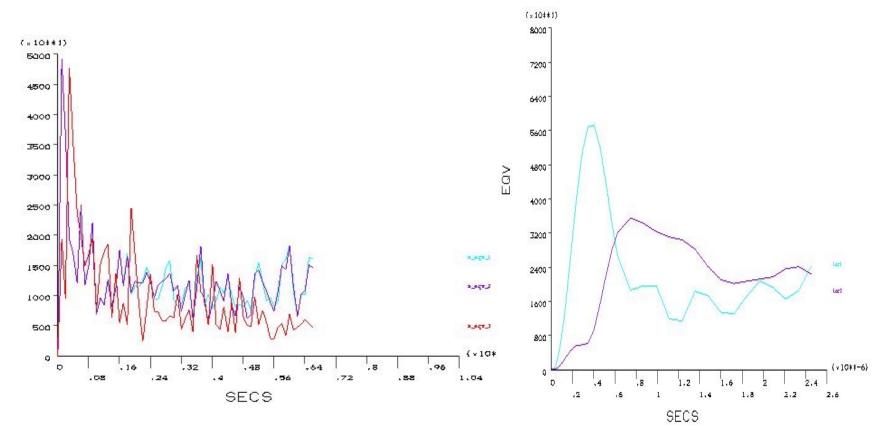
Effect of ratio [pulse/period] on shock stress development



Thermal Shock Analysis of the Exp951 Enclosure Window Von-Mises stress profiles for Inco -718 window and 2-ns pulse length



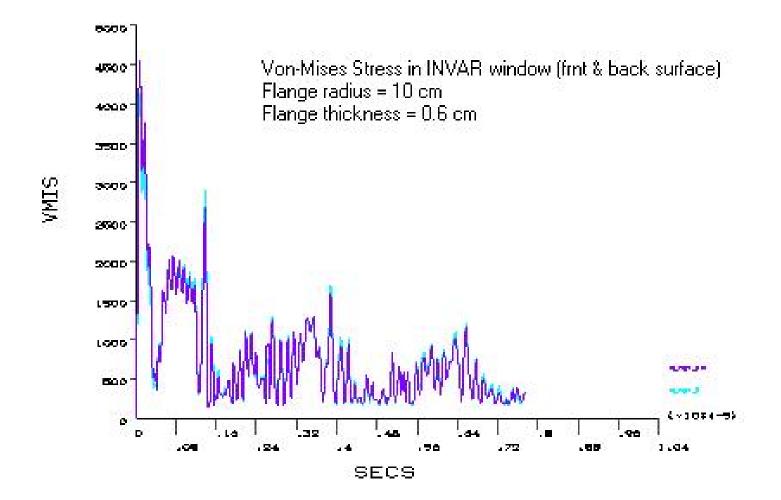
<u>Thermal Shock Analysis</u> Von-Mises stress profile for steel window



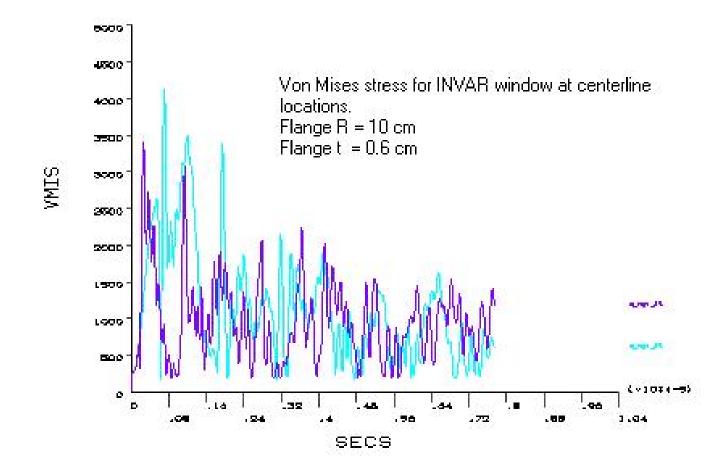
-Mises Stress at locations 1 & 3 following pulse

EQV

Thermal Shock Analysis Von-Mises stress profiles for INVAR window



Thermal Shock Analysis Von-Mises stress profiles for INVAR window



What is Window Failure and how it Impacts on Material Selection & Design

- Vacuum Window Safety Factor dictated by Buckling Failure
 - A safety factor of 4 is typical

• Thermal Shock Failure

- enable material to withstand a single pulse
- design against fatigue failure
- Conservative estimate of trouble is exceeding the yield strength of material
 - for catastrophic failure need to exceed ultimate strength

• Fatigue failure can be short or long-term process

- one can barely overcome single-pulse safety and fatigue failure can arise after just few pulses !
- Through-thickness ringing very important in estimating fatigue due to many cycles of stress it introduces before it dies out

Failure Assessment of E951 Windows A3 Line Aluminum Windows

- Based on 16 TP/24 GeV beam, **1mm RMS sigma** & pulse = 34ns the peak vonMises stresses are for various thicknesses:
 - 3-mil = 303 Mpa
 - 6-mil = 360 Mpa
 - 12-mil = 436 Mpa
 - 24-mil = 368 Mpa
- Aluminum 5052 has Sy = 255 Mpa & Su = 290 Mpa
- Based on latest optics calculations, smallest beam spot in the Aluminum windows of A3 line are much larger than the 1mm RMS sigma providing ample safety factor
- Real beam pulse is triangular (in time space) with larger width (base ~ 100 ns)

Failure Assessment of E951 Windows Experiment Windows

- Based on 16 TP/24 GeV beam, **1mm RMS sigma** & pulse = 34 ns
 - -1-mm (inconel) = 1360 Mpa
 - 2 mm = 1172 Mpa
 - 3-mm = 920 Mpa
 - 6-mm = 736 Mpa (640 MPa for 68 ns pulse !)
 - 12-mm = 860 MPa
- Inconel 718 Strength: Sy = 1034 MPa & Su = 1240 MPa
- Based on latest optics calculations, smallest beam spot at target is smaller than 1mm sigma [0.5mm for x and y]
 - this will leave no safety factor
 - windows may move upstream and downstream
 - re-consider INVAR or Beryllium
- Real beam pulse is triangular (in time space) with larger width (base ~ 100 ns)